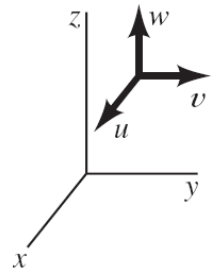


Viscous flow



Equations of motion in cartesian coordinates:

$$\text{x-direction} \quad \rho \left(\underbrace{\frac{\partial u}{\partial t}}_{\text{Local Accel.}} + \underbrace{u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z}}_{\text{Convective Acceleration}} \right) = \underbrace{\rho g_x}_{\text{Gravity}} + \underbrace{\left(\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} \right)}_{\text{Surface forces}}$$

$$\text{y-direction} \quad \rho \left(\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = \rho g_y + \left(\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \tau_{zy}}{\partial z} \right)$$

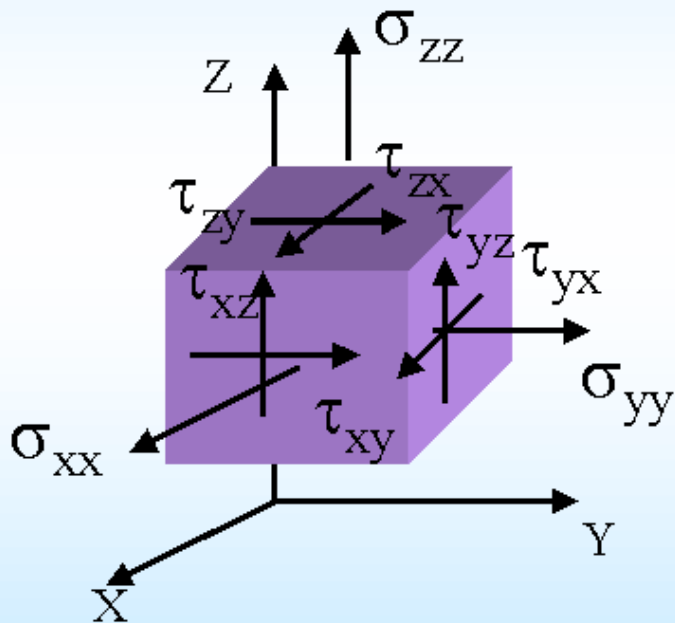
$$\text{z-direction} \quad \rho \left(\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = \rho g_z + \left(\frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_{zz}}{\partial z} \right)$$

The equations of motion include stresses (σ_{ij}) and velocities (u , v and w)

We need a relationship between stresses (σ_{ij}) and velocities (u , v and w)

Stress-deformation relationships

For Newtonian, incompressible fluids, stresses are linearly related to deformations



Normal stresses:

$$\sigma_{xx} = -p + 2\mu \frac{\partial u}{\partial x}$$

$$\sigma_{yy} = -p + 2\mu \frac{\partial v}{\partial y}$$

$$\sigma_{zz} = -p + 2\mu \frac{\partial w}{\partial z}$$

Shearing stresses:

$$\tau_{xy} = \tau_{yx} = \mu \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right)$$

$$\tau_{yz} = \tau_{zy} = \mu \left(\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right)$$

$$\tau_{zx} = \tau_{xz} = \mu \left(\frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} \right)$$

Eliminate stresses

In x-direction: $\rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = \rho g_x + \left(\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} \right)$

$$\sigma_{xx} = -p + 2\mu \frac{\partial u}{\partial x} \longrightarrow \frac{\partial \sigma_{xx}}{\partial x} = -\frac{\partial p}{\partial x} + 2\mu \frac{\partial^2 u}{\partial x^2}$$

$$\tau_{xy} = \tau_{yx} = \mu \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \longrightarrow \frac{\partial \tau_{yx}}{\partial y} = \mu \frac{\partial^2 u}{\partial y^2} + \mu \frac{\partial^2 v}{\partial x \partial y}$$

$$\tau_{zx} = \tau_{xz} = \mu \left(\frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} \right) \longrightarrow \frac{\partial \tau_{zx}}{\partial z} = \mu \frac{\partial^2 w}{\partial x \partial z} + \mu \frac{\partial^2 u}{\partial z^2}$$

The right hand term of the equation becomes:

$$\begin{aligned} \rho g_x + \frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} &= \rho g_x - \frac{\partial p}{\partial x} + 2\mu \frac{\partial^2 u}{\partial x^2} + \mu \frac{\partial^2 u}{\partial x^2} + \mu \frac{\partial^2 u}{\partial x^2} + \mu \frac{\partial v}{\partial x \partial y} + \mu \frac{\partial w}{\partial x \partial z} + \mu \frac{\partial^2 u}{\partial z^2} \\ &= \rho g_x - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right) + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial v}{\partial x \partial y} + \frac{\partial w}{\partial x \partial z} \right) \\ &= \rho g_x - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right) + \underbrace{\mu \frac{\partial}{\partial x} \left(\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \right)}_{=0 : \text{continuity equation}} \\ &= \rho g_x - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right) \end{aligned}$$

Navier-Stokes equations (cartesian)

x-direction: $\rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = - \frac{\partial p}{\partial x} + \rho g_x + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$

$\underbrace{\quad}_{\text{Local Accel.}}$
 $\underbrace{\quad}_{\text{Convective Acceleration}}$
 $\underbrace{\quad}_{\text{Pressure}}$
 $\underbrace{\quad}_{\text{Gravity}}$
 $\underbrace{\quad}_{\text{Viscous forces}}$

y-direction: $\rho \left(\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = - \frac{\partial p}{\partial y} + \rho g_y + \mu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$

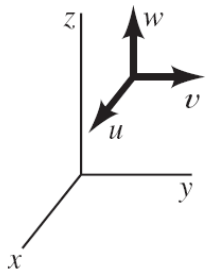
z-direction: $\rho \left(\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = - \frac{\partial p}{\partial z} + \rho g_z + \mu \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$

Momentum equations

+

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

Continuity equation



Stress-deformation relationships for an incompressible fluid (cylindrical)

Normal stresses:

$$\sigma_{rr} = -p + 2\mu \frac{\partial v_r}{\partial r}$$

$$\sigma_{\theta\theta} = -p + 2\mu \left(\frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r}{r} \right)$$

$$\sigma_{zz} = -p + 2\mu \frac{\partial v_z}{\partial z}$$

Shearing stresses:

$$\tau_{r\theta} = \tau_{\theta r} = \mu \left(r \frac{\partial}{\partial r} \left(\frac{v_\theta}{r} \right) + \frac{1}{r} \frac{\partial v_r}{\partial \theta} \right)$$

$$\tau_{\theta z} = \tau_{z\theta} = \mu \left(\frac{\partial v_\theta}{\partial z} + \frac{1}{r} \frac{\partial v_z}{\partial \theta} \right)$$

$$\tau_{rz} = \tau_{zr} = \mu \left(\frac{\partial v_z}{\partial r} + \frac{\partial v_r}{\partial z} \right)$$

Navier-Stokes equations (cylindrical)

r , θ and z momentum equations:

$$r: \underbrace{\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_r}{\partial \theta} - \frac{v_\theta^2}{r} + v_z \frac{\partial v_r}{\partial z} \right)}_{\text{Local Accel.}} = \underbrace{-\frac{\partial p}{\partial r}}_{\text{Pressure}} + \underbrace{\rho g_r}_{\text{Gravity}} + \underbrace{\mu \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_r}{\partial r} \right) - \frac{v_r}{r^2} + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} - \frac{2}{r^2} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial^2 v_r}{\partial z^2} \right)}_{\text{Viscous forces}}$$

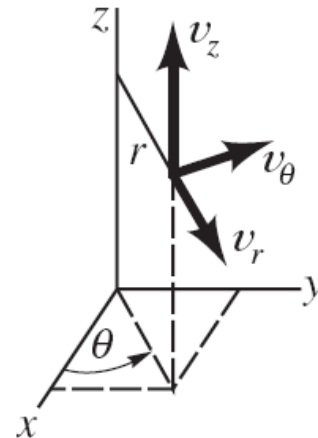
$$\theta: \rho \left(\frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r v_\theta}{r} + v_z \frac{\partial v_\theta}{\partial z} \right) = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \rho g_\theta + \mu \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_\theta}{\partial r} \right) - \frac{v_\theta}{r^2} + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{2}{r^2} \frac{\partial v_r}{\partial \theta} + \frac{\partial^2 v_\theta}{\partial z^2} \right)$$

$$z: \rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \rho g_z + \mu \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right)$$

+

Continuity equation:

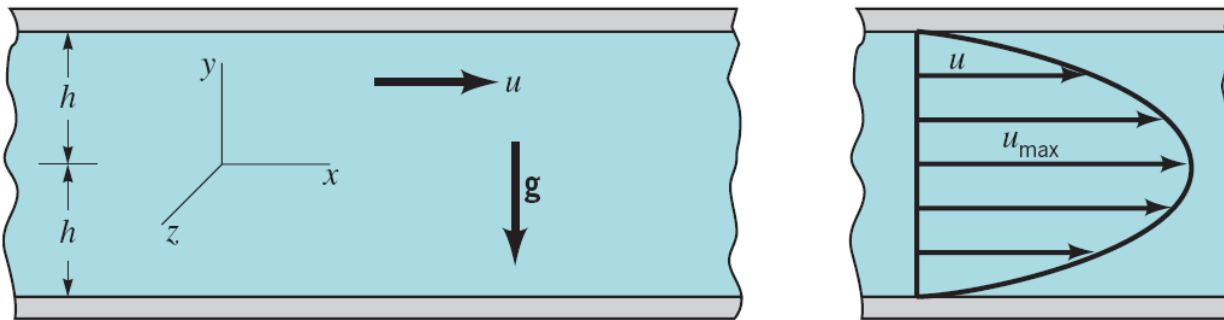
$$\frac{1}{r} \frac{\partial (r v_r)}{\partial r} + \frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial v_z}{\partial z} = 0$$



Simple solutions for viscous, incompressible fluids

- Principal difficulty: nonlinearities from the convective acceleration terms.
- Exact solution exist only for few cases

1) Steady, laminar flow between infinite parallel plates

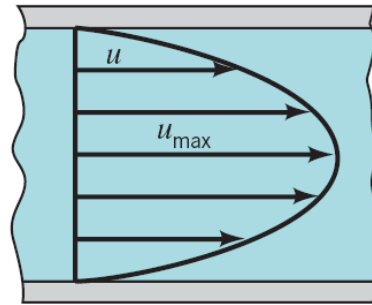
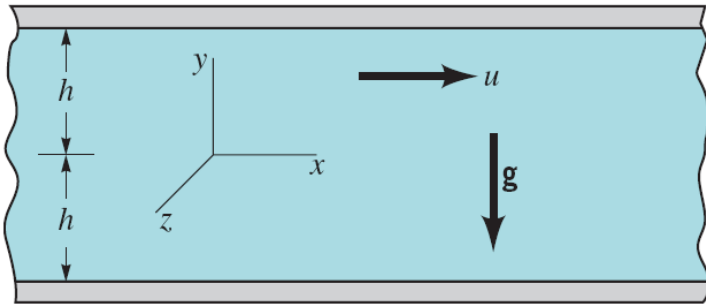


$$\rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = -\frac{\partial p}{\partial x} + \rho g_x + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$

$$\rho \left(\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = -\frac{\partial p}{\partial y} + \rho g_y + \mu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)$$

$$\rho \left(\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = -\frac{\partial p}{\partial z} + \rho g_z + \mu \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$$

Steady, laminar flow between infinite parallel plates (cont'd)



$$x: 0 = -\frac{\partial p}{\partial x} + \mu \frac{\partial^2 u}{\partial y^2} \quad (1)$$

Boundary conditions:

$$u(h) = u(-h) = 0$$

$$y: 0 = -\frac{\partial p}{\partial y} - \rho g \quad (2)$$

$$z: 0 = -\frac{\partial p}{\partial z} \quad (3)$$

$$(2) \Rightarrow p = -\rho g y + f_1(x) \quad (\text{hydrostatic variation in } y)$$

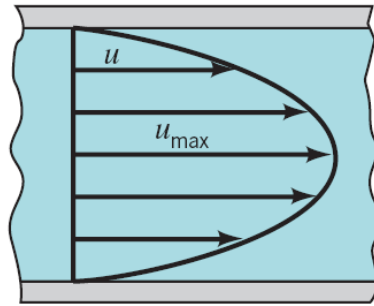
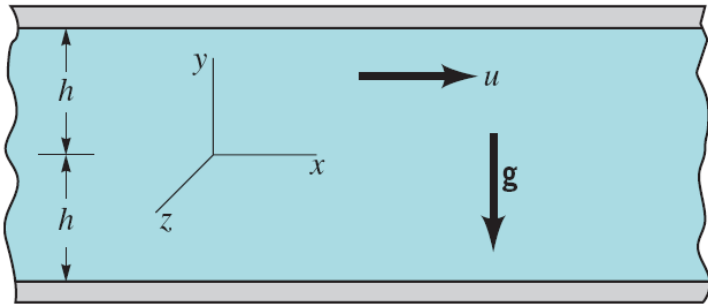
$$(1) \Rightarrow \frac{\partial^2 u}{\partial y^2} = \frac{1}{\mu} \frac{\partial p}{\partial x} \Rightarrow \frac{du}{dy} = \frac{1}{\mu} \frac{\partial p}{\partial x} y + C_1$$

not a function of y

$$\Rightarrow u = \frac{1}{2\mu} \frac{\partial p}{\partial x} y^2 + C_1 y + C_2$$

↑ ↑ from B.C.

Steady, laminar flow between infinite parallel plates (cont'd)



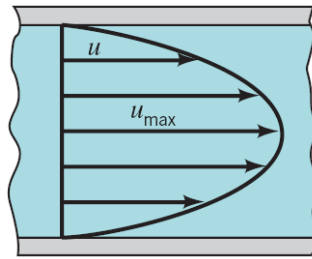
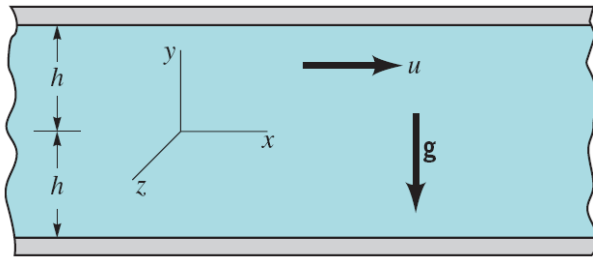
$$u=0 \text{ for } y=h \Rightarrow 0 = \frac{1}{2\mu} \frac{\partial P}{\partial x} h^2 + C_1 h + C_2$$

$$u=0 \text{ for } y=-h \Rightarrow 0 = \frac{1}{2\mu} \frac{\partial P}{\partial x} h^2 - C_1 h + C_2$$

$$\Rightarrow C_1 = 0 \text{ and } C_2 = -\frac{1}{2\mu} \frac{\partial P}{\partial x} h^2$$

$$\therefore \underline{\underline{u = -\frac{1}{2\mu} \frac{\partial P}{\partial x} (h^2 - y^2)}}$$

Steady, laminar flow between infinite parallel plates (cont'd)



Volume rate of flow, q (per unit width in z):

$$q = \int_{-h}^h u \cdot dy = \int_{-h}^h -\frac{1}{2\mu} \frac{\partial P}{\partial x} (h^2 - y^2) dy = -\frac{1}{2\mu} \frac{\partial P}{\partial x} \left[h^2 y - \frac{y^3}{3} \right]_{-h}^h$$

$$= -\frac{1}{2\mu} \frac{\partial P}{\partial x} \left[h^3 - \frac{h^3}{3} - \left(-h^3 + \frac{h^3}{3} \right) \right] = -\frac{1}{2\mu} \frac{\partial P}{\partial x} \left[2h^3 - \frac{2h^3}{3} \right] = -\frac{2h^3}{3\mu} \frac{\partial P}{\partial x}$$

$$\therefore \underline{q = -\frac{2h^3}{3\mu} \frac{\partial P}{\partial x}}$$

The pressure gradient is negative: $-\frac{\partial P}{\partial x} = \frac{\Delta P}{\ell}$

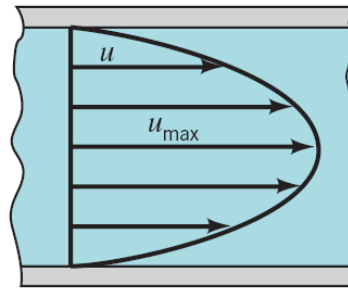
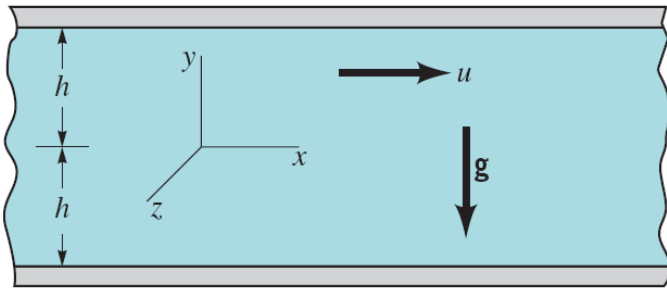
$$\therefore \underline{q = \frac{2h^3}{3\mu} \frac{\Delta P}{\ell}}$$

where $\frac{\Delta P}{\ell}$ is the pressure drop per unit length

Average velocity: $\underline{\underline{\bar{V} = \frac{q}{2h} = \frac{h^2}{3\mu} \frac{\Delta P}{\ell}}}$

Maximum velocity: $\underline{\underline{V_{max} = \frac{1}{2\mu} \frac{\Delta P}{\ell} h^2 = \frac{3}{2} \bar{V}}}$

Steady, laminar flow between infinite parallel plates (cont'd)



For pressures:

$$(2) \Rightarrow P = -\rho g y + f_1(x)$$

$$\Rightarrow \frac{\partial P}{\partial x} = \frac{df_1}{dx} = \text{cte}$$

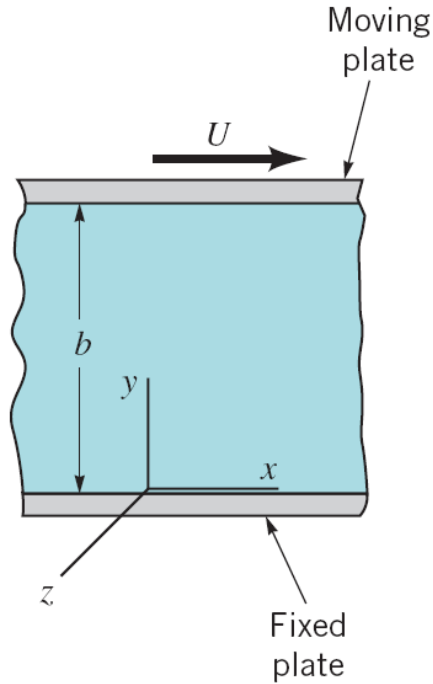
$$\Rightarrow f_1(x) = \frac{\partial P}{\partial x} x + C$$

$$\therefore P(x, y) = -\rho g y + \frac{\partial P}{\partial x} x + C$$

$$\text{Let } P(x=0, y=0) = P_0 \Rightarrow C = P_0 \} \Rightarrow$$

$$\Rightarrow \underline{\underline{P(x, y) = P_0 - \rho g y + \frac{\partial P}{\partial x} x}}$$

Couette flow



The integration of x -momentum yields:

$$u = \frac{1}{2\mu} \frac{\partial P}{\partial x} y^2 + C_1 y + C_2 \quad (1)$$

$$\text{B.C.: } y=0 \rightarrow u=0 \Rightarrow C_2=0$$

$$y=b \rightarrow u=U$$

$$\Rightarrow U = \frac{1}{2\mu} \frac{\partial P}{\partial x} b^2 + C_1 b$$

$$\Rightarrow C_1 = \frac{U}{b} - \frac{1}{2\mu} \frac{\partial P}{\partial x} b \quad (2)$$

$$(1) \Rightarrow u = \frac{1}{2\mu} \frac{\partial P}{\partial x} y^2 + \frac{U}{b} y - \frac{1}{2\mu} \frac{\partial P}{\partial x} b y$$

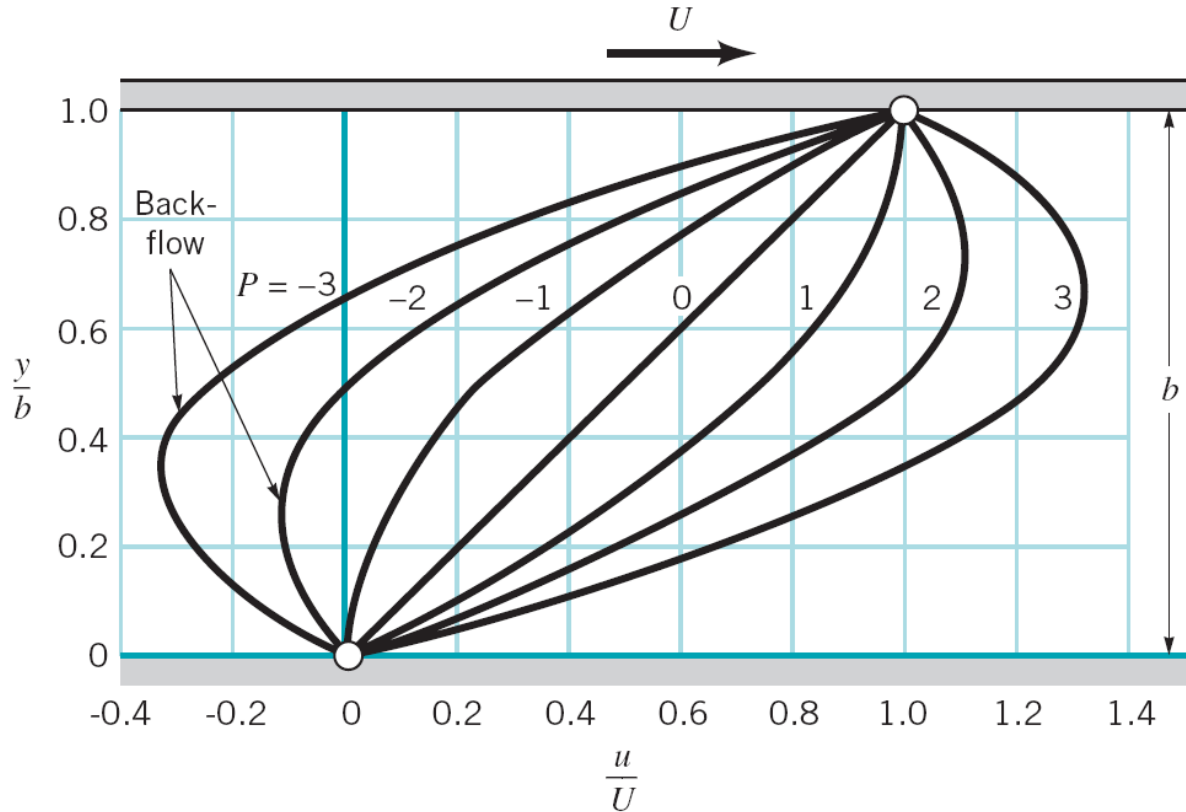
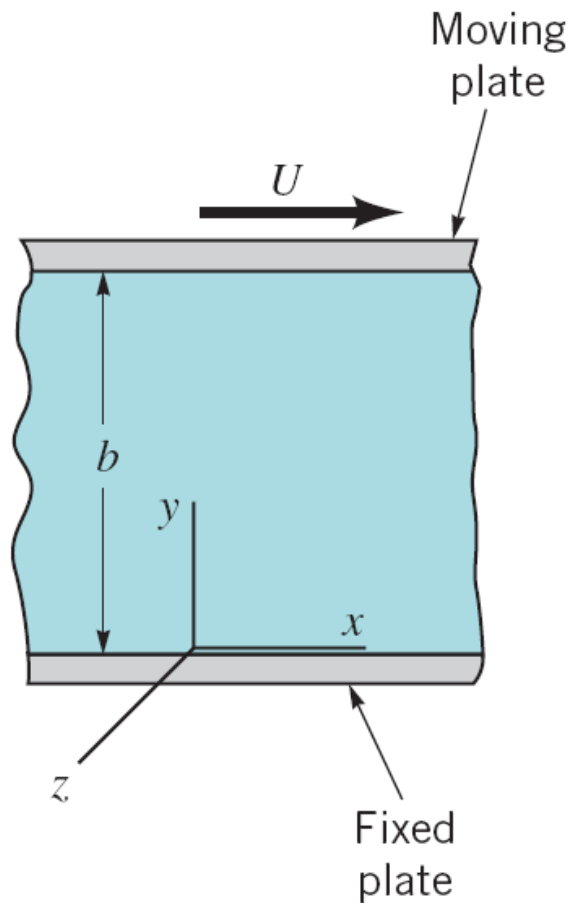
$$\Rightarrow u = \frac{U}{b} y + \frac{1}{2\mu} \frac{\partial P}{\partial x} (y^2 - b y)$$

or, in non-dimensional form:

$$\underline{\underline{\frac{u}{U} = \frac{y}{b} - \frac{b^2}{2\mu U} \frac{\partial P}{\partial x} \frac{y}{b} \left(1 - \frac{y}{b}\right)}}$$

$$\text{Let } P = - \frac{b^2}{2\mu U} \frac{\partial P}{\partial x}$$

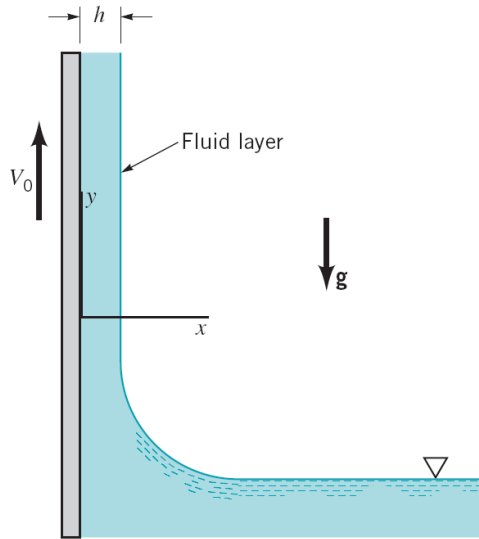
Couette flow



$$\underline{\underline{\frac{u}{U} = \frac{y}{b} - \frac{b^2}{2\mu U} \frac{\partial P}{\partial x} \left(1 - \frac{y}{b}\right)}}$$

$$P = - \frac{b^2}{2\mu U} \frac{\partial P}{\partial x}$$

Example on plane Couette flow



Belt moving vertically upward with constant velocity V_0 , is dragging along a viscous fluid from a reservoir.

Find an expression for the average velocity of the fluid layer.

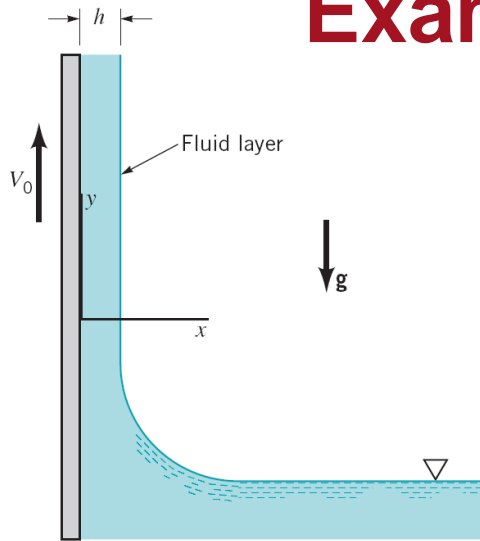
The only velocity component is v ($u = w = 0$)

From continuity: $\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \Rightarrow \frac{\partial v}{\partial y} = 0 \Rightarrow \underline{\underline{v = v(x)}}$

x-momentum: $\frac{\partial P}{\partial x} = 0$
 z-momentum: $\frac{\partial P}{\partial z} = 0$ } \Rightarrow Pressure does not vary in the horizontal plane

Since $P = 0$ at the film interface with air (atmospheric) $\Rightarrow \underline{\underline{P = 0}}$ everywhere in the film

Example on plane Couette flow



y -momentum: $0 = -\rho g + \mu \frac{d^2 v}{dx^2} \Rightarrow \frac{d^2 v}{dx^2} = \frac{\gamma}{\mu} \quad (1)$

Integrate (1) $\Rightarrow \frac{dv}{dx} = \frac{\gamma}{\mu} x + C_1 \quad (2)$

At the interface with air ($x=h$), the shear stress is zero (air drag is negligible)

$$\tau_{x=h} = \mu \left. \frac{dv}{dx} \right|_{x=h} = 0 \xrightarrow{(2)} \frac{\gamma}{\mu} h + C_1 = 0$$

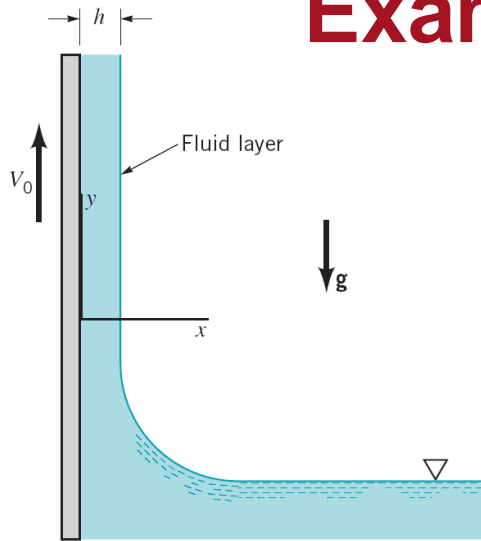
$$\Rightarrow C_1 = -\frac{\gamma h}{\mu} \quad (3)$$

$$\therefore \frac{dv}{dx} = \frac{\gamma}{\mu} x - \frac{\gamma h}{\mu} \Rightarrow v(x) = \frac{1}{2} \frac{\gamma}{\mu} x^2 - \frac{\gamma h}{\mu} x + C_2$$

B.C. $v = V_0$ @ $x=0 \Rightarrow C_2 = V_0$

Hence,
$$v(x) = \frac{\gamma}{2\mu} x^2 - \frac{\gamma h}{\mu} x + V_0 \quad (4)$$

Example on plane Couette flow



The flowrate per unit width, q is:

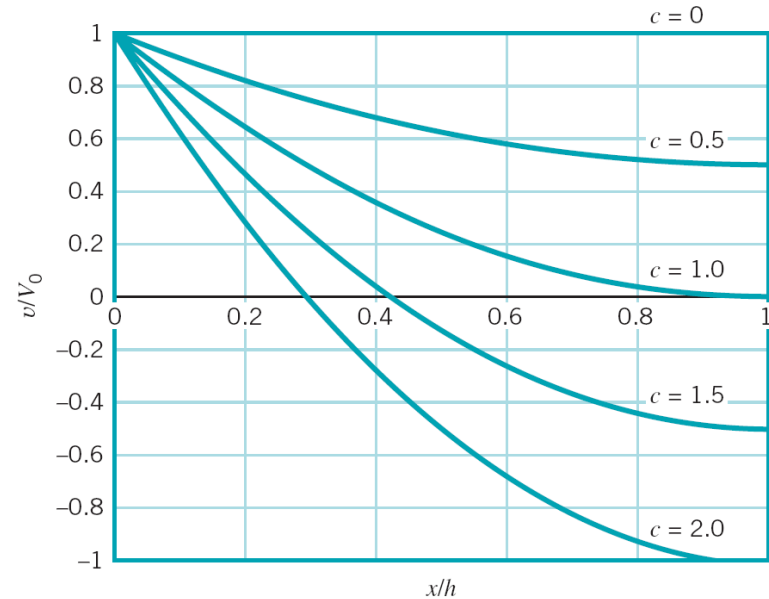
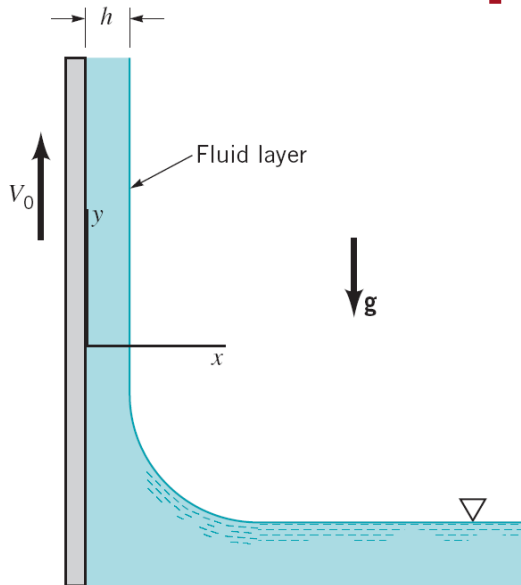
$$q = \int_0^h v(x) dx = \int_0^h \left(\frac{\gamma}{2\mu} x^2 - \frac{\gamma h}{\mu} x + v_0 \right) dx$$
$$= \left[\frac{\gamma x^3}{6\mu} - \frac{\gamma h x^2}{2\mu} + v_0 x \right]_0^h = \frac{\gamma h^3}{6\mu} - \frac{\gamma h^3}{2\mu} + v_0 h$$

$$\Rightarrow \underline{\underline{q = v_0 h - \frac{\gamma h^3}{3\mu}}}$$

The average velocity is:

$$\underline{\underline{\bar{V} = \frac{q}{h} = v_0 - \frac{\gamma h^2}{3\mu}}}$$

Example on plane Couette flow

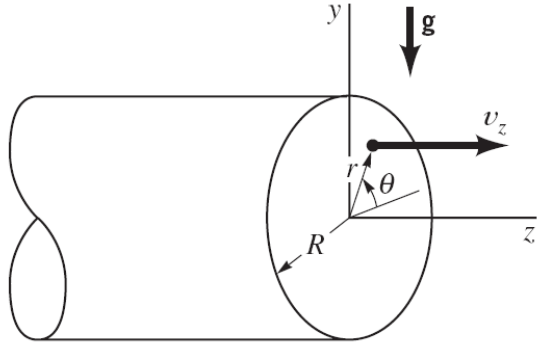


In non-dimensional form:

$$\frac{v}{V_0} = \frac{\gamma h^2}{2\mu V_0} \left(\frac{x}{h}\right)^2 - \frac{\gamma h^2}{\mu V_0} \left(\frac{x}{h}\right) + 1 \quad (5)$$

$$\text{Let } c = \frac{\gamma h^2}{2\mu V_0} \Rightarrow \frac{v}{V_0} = c \left(\frac{x}{h}\right)^2 - 2c \left(\frac{x}{h}\right) + 1 \quad (6)$$

Steady, laminar flow in a horizontal tube: (Hagen-Poiseuille)



Momentum in z: $0 = -\frac{\partial P}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) \right]$

$$\Rightarrow \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) = \frac{1}{\mu} \frac{\partial P}{\partial z} \quad \left(\frac{\partial P}{\partial z} = \text{const} \right)$$

$$\Rightarrow \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) = \frac{1}{\mu} \frac{\partial P}{\partial z} r$$

$$\Rightarrow r \frac{\partial v_z}{\partial r} = \frac{1}{2\mu} \frac{\partial P}{\partial z} r^2 + C_1 \Rightarrow \frac{\partial v_z}{\partial r} = \frac{1}{2\mu} \frac{\partial P}{\partial z} r + \frac{C_1}{r}$$

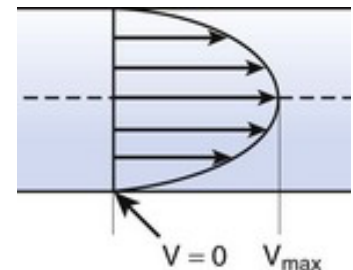
$$\Rightarrow v_z = \frac{1}{4\mu} \frac{\partial P}{\partial z} r^2 + C_1 \ln r + C_2 \quad (1)$$

Boundary conditions:

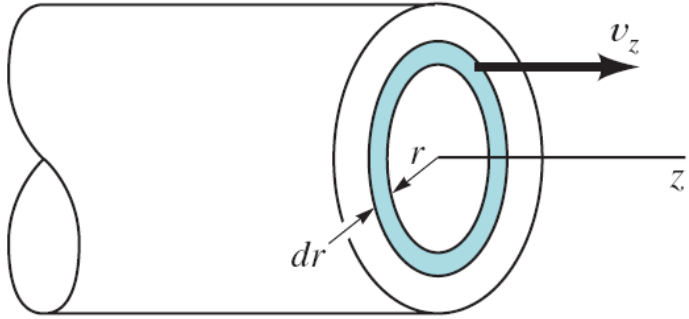
$$v_z \text{ is finite @ } r=0 \Rightarrow C_1 = 0 \quad (2)$$

$$v_z = 0 \text{ @ } r=R \Rightarrow C_2 = -\frac{1}{4\mu} \frac{\partial P}{\partial z} R^2 \quad (3)$$

$$\therefore v_z(r) = -\frac{1}{4\mu} \frac{\partial P}{\partial z} (R^2 - r^2) \quad \text{parabolic profile}$$



Poiseuille's law



$$dQ = v_z dA = v_z (2\pi r \cdot dr)$$

$$Q = \int_0^R v_z 2\pi r dr = -2\pi \frac{1}{4\mu} \frac{\partial P}{\partial z} \int_0^R (R^2 - r^2) r dr$$
$$= -\frac{\pi}{2\mu} \frac{\partial P}{\partial z} \left[\frac{R^2 r^2}{2} - \frac{r^4}{4} \right]_0^R$$

$$= -\frac{\pi}{2\mu} \frac{\partial P}{\partial z} \left[\frac{R^4}{2} - \frac{R^4}{4} \right] = -\frac{\pi R^4}{8\mu} \frac{\partial P}{\partial z}$$

Pressure drop per unit length $\frac{\Delta P}{l} = -\frac{\partial P}{\partial z}$

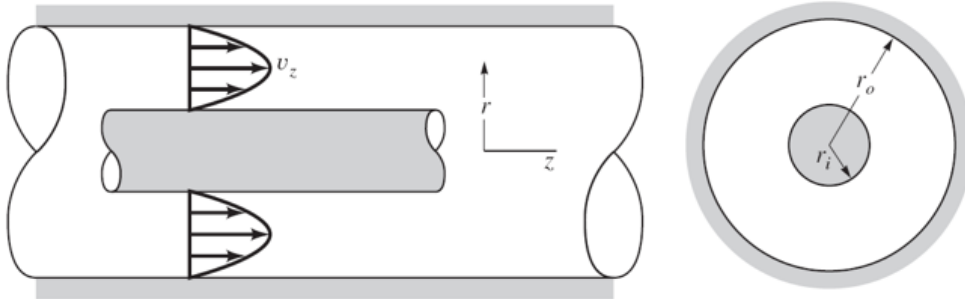
$\therefore Q = \frac{\pi R^4}{8\mu} \frac{\Delta P}{l}$ *Poiseuille's law*

Average velocity: $\bar{V} = \frac{Q}{\pi R^2} = \frac{R^2}{8\mu} \frac{\Delta P}{l}$

Flow is laminar for $Re < 2100$
where $Re = \frac{\rho V D}{\mu}$

$$V_{max} = -\frac{1}{4\mu} \frac{\partial P}{\partial z} R^2 = 2\bar{V}$$

Steady, axial, laminar flow in an annulus



Derive expressions for:

- the axial velocity profile, $u(r)$
- the flow rate, Q

Assuming developed flow, the x-momentum equation of the Navier-Stokes equations is given by:

$$\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) = \frac{1}{\mu} \frac{\partial p}{\partial z}$$

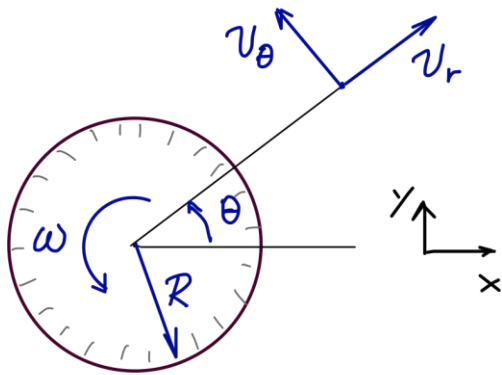
Integrating twice we obtain
$$v_z = \frac{1}{4\mu} \frac{\partial p}{\partial z} r^2 + C_1 \ln r + C_2$$

Applying the boundary conditions $u(r=r_i) = u(r=r_o) = 0$, we come up with the final expression for the velocity distribution:

$$v_z = \frac{1}{4\mu} \left(\frac{\partial p}{\partial z} \right) \left[r^2 - r_o^2 + \frac{r_i^2 - r_o^2}{\ln(r_o/r_i)} \ln \frac{r}{r_o} \right]$$

The flow Q can then be derived by simple integration:
$$Q = \int_{r_i}^{r_o} v_z (2\pi r) dr = -\frac{\pi}{8\mu} \left(\frac{\partial p}{\partial z} \right) \left[r_o^4 - r_i^4 - \frac{(r_o^2 - r_i^2)^2}{\ln(r_o/r_i)} \right]$$

Example: flow around a rotating cylinder



Infinitely long, vertical cylinder turning with rotational speed ω .

Derive an expression for the velocity distribution.

For the particular flow field $v_r = 0$; $v_z = 0$

Continuity in cylindrical coordinates:

$$\frac{1}{r} \frac{\partial(r v_r)}{\partial r} + \frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{\partial v_z}{\partial z} = 0 \Rightarrow \frac{\partial v_\theta}{\partial \theta} = 0$$

The Navier-Stokes equation in the θ -direction:

$$\rho \left(\cancel{\frac{\partial v_\theta}{\partial t}} + \cancel{v_r \frac{\partial v_\theta}{\partial r}} + \cancel{\frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta}} + \cancel{\frac{v_r v_\theta}{r}} + \cancel{v_z \frac{\partial v_\theta}{\partial z}} \right) = -\cancel{\frac{1}{r} \frac{\partial p}{\partial \theta}} + \cancel{\rho g_\theta} + \mu \left(\cancel{\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_\theta}{\partial r} \right)} - \cancel{\frac{v_\theta}{r^2}} + \cancel{\frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2}} + \cancel{\frac{2}{r^2} \frac{\partial v_r}{\partial \theta}} + \cancel{\frac{\partial^2 v_\theta}{\partial z^2}} \right)$$

$$\Rightarrow 0 = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_\theta}{\partial r} \right) - \frac{v_\theta}{r^2} \right]$$

Due to axisymmetry: $\frac{\partial p}{\partial \theta} = 0$

so that
$$\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_\theta}{\partial r} \right) - \frac{v_\theta}{r^2} = 0$$

$$\Rightarrow \frac{\partial^2 v_\theta}{\partial r^2} + \frac{1}{r} \frac{\partial v_\theta}{\partial r} - \frac{v_\theta}{r^2} = 0 \quad (1)$$

Since v_θ is a function of r only:

$$\frac{d^2 v_\theta}{dr^2} + \frac{d}{dr} \left(\frac{v_\theta}{r} \right) = 0 \quad (2)$$

Integrating once, we obtain:

$$\frac{dv_\theta}{dr} + \frac{v_\theta}{r} = C_1 \Rightarrow r \frac{dv_\theta}{dr} + v_\theta = C_1 \cdot r$$

$$\Rightarrow \frac{d(r v_\theta)}{dr} = C_1 \cdot r \Rightarrow r v_\theta = \frac{1}{2} C_1 r^2 + C_2$$

$$\Rightarrow v_{\theta} = \frac{1}{2} C_1 r + \frac{C_2}{r}$$

$$\text{B.C.: } @ r = \infty \quad v_{\theta} = 0 \Rightarrow C_1 = 0$$

$$@ r = R \quad v_{\theta} = \omega R \Rightarrow C_2 = \omega R^2$$

$$\text{Finally: } \underline{\underline{v_{\theta} = \frac{\omega R^2}{r}}}$$